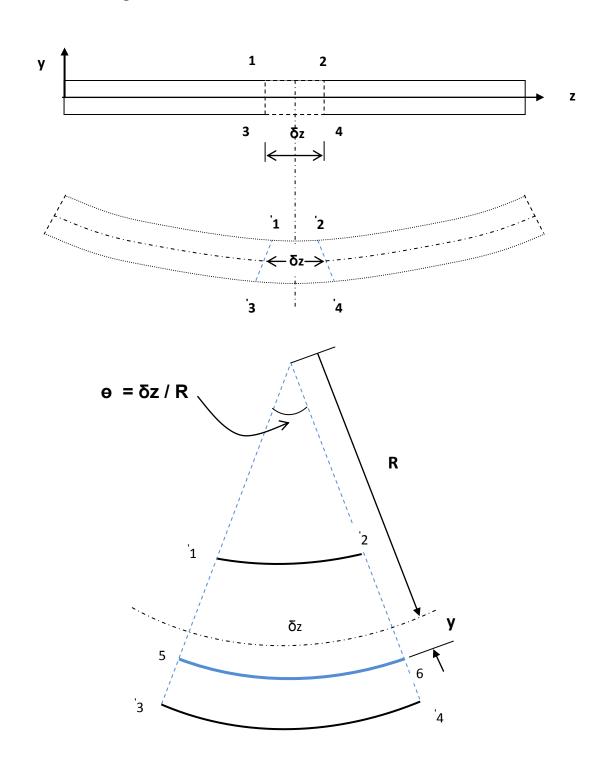
Longitudinal stresses in beams

The figure shown below represents a beam before and after flexural bending :



$$R\theta = \delta z \qquad \rightarrow \rightarrow \rightarrow \theta = \frac{\delta z}{R}$$

Length of
$$(5-6)$$
 fiber = $(R+y)\theta = (R+y)\frac{\delta z}{R}$

Change in length $\Delta L = [Length \ of \ (5-6) - \delta z]$

$$\Delta L = (R + y)\theta - \delta z = (R + y)\frac{\delta z}{R} - \delta z$$

$$\Delta L = \delta z + y\frac{\delta z}{R} - \delta z = y\frac{\delta z}{R}$$

$$\epsilon = \frac{\Delta L}{L} = \frac{y\frac{\delta z}{R}}{\delta z} = \frac{y}{R}$$

$$\sigma = E \in \frac{Ey}{R}$$

But
$$M = \int \sigma. b. y. dy = \int \frac{Ey}{R} b. y. \in dy = \frac{E}{R} \int b y^2 dy = \frac{EI_x}{R}$$

$$\therefore \sigma = \frac{Ey}{R} \longrightarrow \rightarrow \frac{\sigma}{y} = \frac{E}{R}$$

and
$$M = \frac{EI_x}{R} \rightarrow \rightarrow \rightarrow \rightarrow \frac{M}{I_x} = \frac{E}{R}$$

$$\therefore \frac{\sigma}{y} = \frac{M}{I_x} = \frac{E}{R}$$